## Engineering Formula Sheet

## Statistics

## Mean

$\mu=\frac{\sum x_{i}}{n}$
$\mu=$ mean value
$\Sigma \mathrm{x}_{\mathrm{i}}=$ sum of all data values ( $\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}, \ldots$ )
$\mathrm{n}=$ number of data values

## Standard Deviation

$$
\sigma=\sqrt{\frac{\sum\left(x_{i}-\mu\right)^{2}}{n}}
$$

$\sigma=$ standard deviation
$\mathrm{x}_{\mathrm{i}}=$ individual data value $\left(\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}, \ldots\right)$
$\mu=$ mean value
$\mathrm{n}=$ number of data values

## Probability

## Frequency

$\mathrm{f}_{\mathrm{x}}=\frac{\mathrm{n}_{\mathrm{x}}}{\mathrm{n}}$
$P_{x}=\frac{f_{x}}{f_{a}}$
$\mathrm{f}_{\mathrm{x}}=$ relative frequency of outcome x
$\mathrm{n}_{\mathrm{x}}=$ number of events with outcome x
$\mathrm{n}=$ total number of events
$\mathrm{P}_{\mathrm{x}}=$ probability of outcome x
$f_{a}=$ frequency of all events

## Binomial Probability (order doesn't matter)

## Independent Events

$\mathrm{P}(\mathrm{A}$ and B and C$)=\mathrm{P}_{\mathrm{A}} \mathrm{P}_{\mathrm{B}} \mathrm{P}_{\mathrm{C}}$
$P(A$ and $B$ and $C)=$ probability of independent events $A$ and $B$ and $C$ occurring in sequence $P_{A}=$ probability of event $A$

## Mutually Exclusive Events

$P(A$ or $B)=P_{A}+P_{B}$
$P(A$ or $B)=$ probability of either mutually exclusive
event A or B occurring in a trial
$\mathrm{P}_{\mathrm{A}}=$ probability of event A
$\Sigma x_{i}=$ sum of all data values ( $x_{1}, x_{2}, x_{3}, \ldots$ )
$P_{k}=\frac{n!\left(p^{k}\right)\left(q^{n-k}\right)}{k!(n-k)!}$
$P_{k}=$ binomial probability of $k$ successes in $n$ trials
$\mathrm{p}=$ probability of a success
$q=1-p=$ probability of failure
$k=$ number of successes
$\mathrm{n}=$ number of trials

## Mode

Place data in ascending order.
Mode = most frequently occurring value
If two values occur at the maximum frequency the data set is bimodal.
If three or more values occur at the maximum frequency the data set is multi-modal.

## Median

Place data in ascending order.
If n is odd, median = central value
If n is even, median = mean of two central values
$\mathrm{n}=$ number of data values

## Range

Range $=x_{\text {max }}-x_{\text {min }}$
$x_{\text {max }}=$ maximum data value
$x_{\text {min }}=$ minimum data value
$\mathrm{n}=$ number of data values

## Conditional Probability

$P(A \mid D)=\frac{P(A) \cdot P(D \mid A)}{P(A) \cdot P(D \mid A)+P(\sim A) \cdot P(D \mid \sim A)}$
$P(A \mid D)=$ probability of event $A$ given event $D$
$P(A)=$ probability of event $A$ occurring
$P(\sim A)=$ probability of event $A$ not occurring
$P(D \nmid \sim A)=$ probability of event $D$ given event $A$ did not occur

## Plane Geometry



## Triangle

Area $=1 / 2$ bh
$a^{2}=b^{2}+c^{2}-2 b c \cdot \cos \angle A$
$\mathrm{b}^{2}=\mathrm{a}^{2}+\mathrm{c}^{2}-2 \mathrm{ac} \cdot \cos \angle \mathrm{B}$
$c^{2}=a^{2}+b^{2}-2 a b \cdot \cos \angle C$


## Rectangle

Perimeter $=2 a+2 b$
Area $=\mathrm{ab}$



## Regular Polygons

Area $=\mathrm{n} \frac{\mathrm{s}\left(\frac{1}{2} \mathrm{f}\right)}{2}$

$\mathrm{n}=$ number of sides

## Trapezoid

Area $=1 / 2(a+b) h$


## Sphere

Volume $\frac{4}{3} \pi r^{3}$
Surface Area $=4 \pi r^{2}$


Volume $=\frac{\pi r^{2} h}{3}$
Surface Area $=\pi r \sqrt{r^{2}+h^{2}}$


## Right Circular Cone



## Pyramid

Volume $=\frac{A h}{3}$
A = area of base


| Irregular Prism <br> Volume $=\mathrm{Ah}$ <br> A $=$ area of base <br> Constants <br> $\mathrm{g}=9.8 \mathrm{~m} / \mathrm{s}^{2}=32.27 \mathrm{ft} / \mathrm{s}^{2}$ <br> $\mathrm{G}=6.67 \times 10^{-11} \mathrm{~m}^{3} / \mathrm{kg} \cdot \mathrm{s}^{2}$ <br> $\pi=3.14159$ |
| :--- |

## Conversions

| Mass |  |
| :--- | :--- |
| 1 kg | $=2.205 \mathrm{lb}_{\mathrm{m}}$ |
| 1 slug | $=32.2 \mathrm{lb}_{\mathrm{m}}$ |
| 1 ton | $=2000 \mathrm{lb}_{\mathrm{m}}$ |

## Length

| 1 m | $=3.28 \mathrm{ft}$ |
| :--- | :--- |
| 1 km | $=0.621 \mathrm{mi}$ |
| 1 in. | $=2.54 \mathrm{~cm}$ |
| 1 mi | $=5280 \mathrm{ft}$ |
| 1 yd | $=3 \mathrm{ft}$ |

## Temperature Change

$1 \mathrm{~K}=1{ }^{\circ} \mathrm{C}$
$=1.8{ }^{\circ} \mathrm{F}$
$=1.8{ }^{\circ} \mathrm{R}$

| Area |  |
| ---: | :--- |
| 1 acre | $=4047 \mathrm{~m}^{2}$ |
|  | $=43,560 \mathrm{ft}^{2}$ |
|  | $=0.00156 \mathrm{mi}^{2}$ |


| Force |  |
| :--- | :--- |
| 1 N | $=0.225 \mathrm{lb}_{\mathrm{f}}$ |
| 1 kip | $=1,000 \mathrm{lb}_{\mathrm{f}}$ |

## Pressure

## Volume

$1 \mathrm{~L} \quad=0.264 \mathrm{gal}$
$=0.0353 \mathrm{ft}^{3}$
$=33.8 \mathrm{fl} \mathrm{oz}$
$1 \mathrm{~mL}=1 \mathrm{~cm}^{3}=1 \mathrm{cc}$

## Time

$1 \mathrm{~d}=24 h$
$1 \mathrm{~h}=60 \mathrm{~min}$
$1 \mathrm{~min}=60 \mathrm{~s}$
$1 \mathrm{yr}=365 \mathrm{~d}$

## Energy

$$
\begin{aligned}
1 \mathrm{~J} \quad & =0.239 \mathrm{cal} \\
& =9.48 \times 10^{-4} \mathrm{Btu} \\
& =0.7376 \mathrm{ft} \cdot \mathrm{~b}_{\mathrm{f}} \\
1 \mathrm{~kW} \mathrm{~h} & =3,6000,000 \mathrm{~J}
\end{aligned}
$$

## Defined Units

$1 \mathrm{~J}=1 \mathrm{~N} \cdot \mathrm{~m}$
$1 \mathrm{~N}=1 \mathrm{~kg} \cdot \mathrm{~m} / \mathrm{s}^{2}$
$1 \mathrm{~Pa}=1 \mathrm{~N} / \mathrm{m}^{2}$
$1 \mathrm{~V}=1 \mathrm{~W} / \mathrm{A}$
$1 \mathrm{~W}=1 \mathrm{~J} / \mathrm{s}$
$1 \mathrm{~W}=1 \mathrm{~V} / \mathrm{A}$
$1 \mathrm{~Hz}=1 \mathrm{~s}^{-1}$
$1 \mathrm{~F}=1 \mathrm{~A} \cdot \mathrm{~s} / \mathrm{V}$
$1 \mathrm{H}=1 \mathrm{~V} \cdot \mathrm{~s} / \mathrm{V}$

## SI Prefixes

| Numbers Less Than One |  |  |
| :---: | :---: | :---: |
| Power of 10 | Prefix | Abbreviation |
| $10^{-1}$ | deci- | d |
| $10^{-2}$ | centi- | c |
| $10^{-3}$ | milli- | m |
| $10^{-6}$ | micro- | $\mu$ |
| $10^{-9}$ | nano- | n |
| $10^{-12}$ | pico- | p |
| $10^{-15}$ | femto- | f |
| $10^{-18}$ | atto- | a |
| $10^{-21}$ | zepto- | z |
| $10^{-24}$ | yocto- | y |$\quad$| Numbers Greater Than One |  |  |  |
| :---: | :---: | :---: | :---: |
|  | Power of 10 | Prefix | Abbreviation |
|  | $10^{1}$ | deca- | da |
|  | $10^{2}$ | hecto- | h |
|  | $10^{3}$ | kilo- | k |
|  | $10^{6}$ | Mega- | M |
|  | $10^{9}$ | Giga- | G |
|  | $10^{12}$ | Tera- | T |
|  | $10^{15}$ | Peta- | P |
|  | $10^{21}$ | Zetta- | Z |
| $10^{24}$ | Yotta- | Y |  |

Equations

| Mass and Weight |
| :--- |
| $M=V D_{m}$ |
| $W=m g$ |
| $W=V D_{w}$ |
| $V=$ volume |
| $D_{m}=$ mass density |
| $m=$ mass |
| $D_{w}=$ weight density |
| $g=$ acceleration due to gravity |

## Temperature

$\mathrm{T}_{\mathrm{K}}=\mathrm{T}_{\mathrm{C}}+273$
$T_{R}=T_{F}+460$
$\frac{T_{F}-32}{180}=\frac{T_{C}}{100}$
$\mathrm{T}_{\mathrm{K}}=$ temperature in Kelvin
$\mathrm{T}_{\mathrm{C}}=$ temperature in Celsius
$\mathrm{T}_{\mathrm{R}}=$ temperature in Rankin
$\mathrm{T}_{\mathrm{F}}=$ temperature in Fahrenheit

Force
$\mathrm{F}=\mathrm{ma}$
F = force
$\mathrm{m}=$ mass
$\mathrm{a}=$ acceleration

## Equations of Static Equilibrium

$\Sigma \mathrm{F}_{\mathrm{x}}=0 \quad \Sigma \mathrm{~F}_{\mathrm{y}}=0 \quad \Sigma \mathrm{M}_{\mathrm{p}}=0$
$\mathrm{F}_{\mathrm{x}}=$ force in the x -direction
$F_{y}=$ force in the $y$-direction
$M_{P}=$ moment about point $P$

## Equations (Continued)

| Energy: Work |
| :--- |
| $W=\mathrm{F} \cdot \mathrm{d}$ |
| $\mathrm{W}=$ work |
| $\mathrm{F}=$ force |
| $\mathrm{d}=$ distance |

## Power

$P=\frac{E}{t}=\frac{W}{t}$
$P=\frac{\tau \cdot r p m}{5252}$
$\mathrm{P}=$ power
$E=$ energy
W = work
$\mathrm{t}=$ time
$\tau=$ torque
rpm = revolutions per minute

## Efficiency

Efficiency (\%) $=\frac{P_{\text {out }}}{P_{\text {in }}} \cdot 100 \%$
$\mathrm{P}_{\text {out }}=$ useful power output
$P_{\text {in }}=$ total power input

| Energy: Potential |
| :--- |
| $\mathrm{U}=\mathrm{mgh}$ |
| $\mathrm{U}=$ potential energy |
| $\mathrm{m}=$ mass |
| $\mathrm{g}=$ acceleration due to gravity |
| $\mathrm{h}=$ height |


| Energy: Kinetic |
| :--- |
| $\mathrm{K}=\frac{1}{2} \mathrm{mv}^{2}$ |
| $\mathrm{~K}=$ kinetic energy |
| $\mathrm{m}=$ mass |
| $\mathrm{V}=$ velocity |


| Energy: Thermal |
| :--- |
| $\mathrm{Q}=\mathrm{mc} \Delta \mathrm{T}$ |
| $\mathrm{Q}=$ thermal energy |
| $\mathrm{m}=$ mass |
| $\mathrm{C}=$ specific heat |
| $\Delta \mathrm{T}=$ change in temperature |

## Fluid Mechanics

$P=\frac{F}{A}$
$\frac{\mathrm{V}_{1}}{\mathrm{~T}_{1}}=\frac{\mathrm{V}_{2}}{\mathrm{~T}_{2}}$ (Charles' Law)
$\frac{\mathrm{P}_{1}}{\mathrm{~T}_{1}}=\frac{\mathrm{P}_{2}}{\mathrm{~T}_{2}}$ (Guy-Lussanc's Law)
$\mathrm{P}_{1} \mathrm{~V}_{1}=\mathrm{P}_{2} \mathrm{~V}_{2}$ (Boyle's Law)
$Q=A v$
$\mathrm{A}_{1} \mathrm{v}_{1}=\mathrm{A}_{2} \mathrm{v}_{2}$
Horsepower $=\frac{\text { QP }}{1714}$
absolute pressure = gauge pressure + atmospheric pressure
$\mathrm{P}=$ absolute pressure
$\mathrm{F}=\mathrm{Force}$
A = Area
$\mathrm{V}=$ volume
$\mathrm{T}=$ absolute temperature
$Q$ = flow rate
v = flow velocity

| Mechanics |
| :---: |
| $\mathrm{s}=\frac{\mathrm{d}}{\mathrm{t}} \quad($ where acceleration $=0)$ |
| $\mathbf{v}=\frac{\mathbf{d}}{\mathrm{t}}($ where acceleration $=0)$ |
| $a=\frac{v_{f}-v_{i}}{t}$ |
| $X=\underline{v_{i} \sin (2 \theta)}$ |
| -g |
| $v=v_{0}+$ at |
| $d=d_{0}+v_{0} t+1 / 2 a t^{2}$ |
| $\mathrm{v}^{2}=\mathrm{v}_{0}{ }^{2}+2 \mathrm{a}\left(\mathrm{d}-\mathrm{d}_{0}\right)$ |
| $\boldsymbol{\tau}=\mathrm{dF} \sin \theta$ |
| s = speed |
| $\mathrm{v}=$ velocity |
| $\mathrm{a}=$ acceleration |
| $\mathrm{X}=$ range |
| $\mathrm{t}=\text { time }$ |
| $\begin{aligned} & g=\text { acceleration due to gravity } \\ & d=\text { distance } \end{aligned}$ |
| $\theta=$ angle |
| $\tau=$ torque |
| $F=$ force |

## Electricity

> Ohm's Law
> $\mathrm{V}=\mathrm{IR}$
> $\mathrm{P}=\mathrm{IV}$
> $\mathrm{R}_{\mathrm{T}}$ (series) $=\mathrm{R}_{1}+\mathrm{R}_{2}+\cdots+\mathrm{R}_{\mathrm{n}}$
> $\mathrm{R}_{\mathrm{T}}$ (parallel) $=\frac{1}{\frac{1}{R_{1}}+\frac{1}{R_{2}}+\cdots+\frac{1}{R_{n}}}$

## Kirchhoff's Current Law

$I_{T}=I_{1}+I_{2}+\cdots+I_{n}$

$$
\text { or } \mathrm{I}_{\mathrm{T}}=\sum_{\mathrm{k}=1}^{\mathrm{n}} \mathrm{I}_{\mathrm{k}}
$$

## Kirchhoff's Voltage Law

$\mathrm{V}_{\mathrm{T}}=\mathrm{V}_{1}+\mathrm{V}_{2}+\cdots+\mathrm{V}_{\mathrm{n}}$

$$
\text { or } \quad V_{T}=\sum_{k=1}^{n} V_{k}
$$

$\mathrm{V}=$ voltage
$\mathrm{V}_{\mathrm{T}}=$ total voltage
I = current
$\mathrm{I}_{\mathrm{T}}=$ total current
$R=$ resistance
$R_{T}=$ total resistance
$\mathrm{P}=$ power

## Thermodynamics

$P=Q^{\prime}=A U \Delta T$
$P=\frac{Q}{\Delta t}$
$U=\frac{1}{R}=\frac{k}{L}$
$P=\frac{k A \Delta T}{L}$
$\mathrm{A}_{1} \mathrm{v}_{1}=\mathrm{A}_{2} \mathrm{~V}_{2}$
$\mathrm{P}_{\text {net }}=\sigma \operatorname{Ae}\left(\mathrm{T}_{2}{ }^{4}-\mathrm{T}_{1}{ }^{4}\right)$
$\mathrm{P}=$ rate of heat transfer
$Q=$ thermal energy
A = Area of thermal conductivity
$U=$ coefficient of heat conductivity (U-factor)
$\Delta T=$ change in temperature
$\Delta t=$ change in time
$R=$ resistance to heat flow ( $R$-value)
$\mathrm{k}=$ thermal conductivity
$\mathrm{v}=$ velocity
$P_{\text {net }}=$ net power radiated
$\sigma=5.6696 \times 10^{-8} \frac{\mathrm{~W}}{\mathrm{~m}^{2} \cdot \mathrm{k}^{4}}$
$e=$ emissivity constant

## Section Properties

## Moment of Inertia <br> $I_{x x}=\frac{b h^{3}}{12}$ <br> 

$\mathrm{I}_{\mathrm{xx}}=$ moment of inertia of a rectangular section about $x$ - $x$ axis

## Complex Shapes Centroid

$\bar{x}=\frac{\sum x_{i} A_{i}}{\sum A_{i}}$ and $\bar{y}=\frac{\sum y_{i} A_{i}}{\sum A_{i}}$
$\bar{x}=x$-distance to the centroid
$\bar{y}=y$-distance to the centroid
$x_{i}=x$ distance to centroid of shape $i$
$y_{i}=y$ distance to centroid of shape $i$
$\mathrm{A}_{\mathrm{i}}=$ Area of shape i

| Material Properties |
| :--- |
| Stress (axial) <br> $\sigma=\frac{\mathrm{F}}{\mathrm{A}}$ <br> $\sigma=$ stress <br> $\mathrm{F}=$ axial force <br> $\mathrm{A}=$ cross-sectional area |

## Strain (axial)

$\epsilon=\frac{\delta}{\mathrm{L}_{0}}$
$\epsilon=$ strain
$L_{0}=$ original length
$\delta=$ change in length

## Modulus of Elasticity

$\mathrm{E}=\frac{\sigma}{\varepsilon}$
$\mathrm{E}=\frac{\sigma\left(\mathrm{F}_{2}-\mathrm{F}_{1}\right) \mathrm{L}_{0}}{\left(\delta_{2}-\delta_{1}\right) \mathrm{A}}$
$\mathrm{E}=$ modulus of elasticity
$\sigma=$ stress
$\varepsilon=$ strain
$\mathrm{A}=$ cross-sectional area
$\mathrm{F}=$ axial force
$\delta=$ deformation

## Deformation: Axial

$\delta=\frac{\mathrm{FL}_{0}}{\mathrm{AE}}$
$\delta=$ deformation
F = axial force
$\mathrm{L}_{0}=$ original length

## Rectangle Centroid

$\bar{x}=\frac{b}{2}$ and $\bar{y}=\frac{h}{2}$


Right Triangle Centroid
$\bar{x}=\frac{b}{3}$ and $\bar{y}=\frac{h}{3}$


## Semi-circle Centroid

$\bar{x}=r$ and $\bar{y}=\frac{4 r}{3 \pi}$

$\bar{x}=x$-distance to the centroid
$\bar{y}=y$-distance to the centroid

## Structural Analysis

| Beam Formulas |  |  |
| :---: | :---: | :---: |
|  | Reaction <br> Moment <br> Deflection | $\begin{gathered} R_{A}=R_{B}=\frac{P}{2} \\ \mathrm{M}_{\max }=\frac{\mathrm{PL}}{4} \text { (at point of load) } \\ \Delta_{\max }=\frac{\mathrm{PL}}{48 \mathrm{E}} \text { (at point of load) } \end{gathered}$ |
|  | Reaction <br> Moment <br> Deflection | $\begin{aligned} & R_{A}=R_{B}=\frac{\omega L}{2} \\ & M_{\max }=\frac{\omega L^{2}}{8} \quad \text { (at center) } \\ & \Delta_{\max }=\frac{55 L^{4}}{384 E I} \quad \text { (at center) } \end{aligned}$ |
|  | Reaction Moment <br> Deflection | $\begin{aligned} & R_{A}=R_{B}=P \\ & M_{\max }=P a \quad \text { (between loads) } \\ & \Delta_{\max }=\frac{P a}{24 E I}\left(3 L^{2}-4 a^{2}\right) \quad \text { (at center) } \end{aligned}$ |
|  | Reaction <br> Moment <br> Deflection | $\begin{aligned} & R_{A}=\frac{P b}{L} \text { and } R_{B}=\frac{P a}{L} \\ & M_{\max }=\frac{\mathrm{Pab}}{L}(\text { at Point of Load }) \\ & \Delta_{\max }=\frac{\mathrm{Pab}(a+2 b \sqrt{3 a(a+2 b)}}{27 \mathrm{EL}} \\ & \left(\text { at } \mathrm{x}=\sqrt{\frac{a(a+2 b)}{3,}} \text { when } \mathrm{a}>\mathrm{b}\right) \end{aligned}$ |

$\mathrm{A}=$ cross-sectional area
$\mathrm{E}=$ modulus of elasticity

## Truss Analysis

$2 J=M+R$
$J=$ number of joints
M =number of members
$R=$ number of reaction forces

## Simple Machines

## Mechanical Advantage (MA)




Wheel and Axle


Effort at Wheel


## Pulley Systems

IMA = Total number of strands of a single string supporting the resistance

IMA $=\frac{D_{E}(\text { string pulled })}{D_{R}(\text { resistance lifted })}$


## Wedge

IMA $=\frac{L(\perp \text { to height })}{H}$


## Screw



$$
\text { Pitch }=\frac{1}{\mathrm{TPI}}
$$



Pitch
$C=$ Circumference
$r=$ radius
Pitch = distance between threads
TPI = Threads Per Inch

## Compound Machines

$M A_{\text {TOTAL }}=\left(M A_{1}\right)\left(M A_{2}\right)\left(M A_{3}\right) \ldots$

Gears; Sprockets with Chains; and Pulleys with Belts Ratios

$$
\begin{aligned}
& G R=\frac{N_{\text {out }}}{N_{\text {in }}}=\frac{d_{\text {out }}}{d_{\text {in }}}=\frac{\omega_{\text {in }}}{\omega_{\text {out }}}=\frac{T_{\text {out }}}{T_{\text {in }}} \\
& \frac{d_{\text {out }}}{d_{\text {in }}}=\frac{\omega_{\text {in }}}{\omega_{\text {out }}}=\frac{T_{\text {out }}}{T_{\text {in }}} \text { (pulleys) }
\end{aligned}
$$

## Compound Gears

$\mathrm{GR}_{\text {TOTAL }}=\left(\frac{\mathrm{B}}{\mathrm{A}}\right)\left(\frac{\mathrm{D}}{\mathrm{C}}\right)$


[^0]
## Structural Design

| Steel Beam Design: Shear |
| :--- |
| $\mathrm{V}_{\mathrm{a}}=\frac{\mathrm{V}_{\mathrm{n}}}{\Omega_{\mathrm{v}}}$ |
| $\mathrm{V}_{\mathrm{n}}=0.6 \mathrm{~F}_{\mathrm{y}} \mathrm{A}_{\mathrm{w}}$ |
| $\mathrm{V}_{\mathrm{a}}=$ allowable shear strength |
| $\mathrm{V}_{\mathrm{n}}=$ nominal shear strength |
| $\Omega_{\mathrm{v}}=1.5=$ factor of safety for shear |
| $\mathrm{F}_{\mathrm{y}}=$ yield stress |
| $\mathrm{A}_{\mathrm{w}}=$ area of web |

## Storm Water Runoff

## Storm Water Drainage

$\mathrm{Q}=\mathrm{C}_{\mathrm{f}} \mathrm{CiA}$
$C_{c}=\frac{C_{1} A_{1}+C_{2} A_{2}+\cdots}{A_{1}+A_{2}+\cdots}$
$\mathrm{Q}=$ peak storm water runoff rate ( $\mathrm{ft}{ }^{3} / \mathrm{s}$ )
$\mathrm{C}_{\mathrm{f}}=$ runoff coefficient adjustment factor
$\mathrm{C}=$ runoff coefficient
$\mathrm{i}=$ rainfall intensity (in./h)
A = drainage area (acres)

| Runoff Coefficient Adjustment Factor | fficient Factor |
| :---: | :---: |
| Return Period | Cf |
| 1, 2, 5, 10 | 1.0 |
| 25 | 1.1 |
| 50 | 1.2 |
| 100 | 1.25 |
| Water Supply |  |
| Hazen-Williams Formula |  |
| $\mathrm{h}_{\mathrm{f}}=\frac{10.44 \mathrm{LQ}^{1.85}}{\mathrm{C}^{1.85} \mathrm{~d}^{4.8655}}$ |  |
| $\mathrm{h}_{\mathrm{f}}=$ head loss due to <br> $\mathrm{L}=$ length of pipe ( ft ) <br> $\mathrm{Q}=$ water flow rate <br> C = Hazen-Williams <br> $\mathrm{d}=$ diameter of pipe | friction ( $\mathrm{ft} \mathrm{of} \mathrm{H}_{2} \mathrm{O}$ ) <br> gpm) <br> constant <br> (in.) |

## Dynamic Head

dynamic head = static head - head loss

## Steel Beam Design: Moment

$M_{a}=\frac{M_{n}}{\Omega_{\mathrm{b}}}$
$M_{n}=F_{y} Z_{x}$
$\mathrm{M}_{\mathrm{a}}=$ allowable bending moment
$\mathrm{M}_{\mathrm{n}}=$ nominal moment strength
$\Omega_{\mathrm{b}}=1.67=$ factor of safety for bending moment
$\mathrm{F}_{\mathrm{y}}=$ yield stress
$\mathrm{Z}_{\mathrm{x}}=$ plastic section modulus about neutral axis

| Rational Method Runoff Coefficients |
| :--- |
| Categorized by Surface |
| Foreser |

## Spread Footing Design

$q_{\text {net }}=q_{\text {allowable }}-p_{\text {footing }}$
$\mathrm{p}_{\text {footing }}=\mathrm{t}_{\text {footing }} \cdot 150 \frac{\mathrm{lb}}{\mathrm{ft}^{2}}$
$q=\frac{P}{A}$
$\mathrm{q}_{\text {net }}=$ net allowable soil bearing pressure
$q_{\text {allowable }}=$ total allowable soil bearing pressure
$\mathrm{p}_{\text {footing }}=$ soil bearing pressure due to footing weight $\mathrm{t}_{\text {footing }}=$ thickness of footing $\mathrm{q}=$ soil bearing pressure
$\mathrm{P}=$ column load applied
A = area of footing
Hazen-Williams Constants

| Pipe Material | Typical Range | Clean, New Pipe | Typical Design <br> Value |
| :---: | :---: | :---: | :---: |
| Cast Iron and <br> Wrought Iron | $80-150$ | 130 | 100 |
| Copper, Glass <br> or Brass | $120-150$ | 140 | 130 |
| Cement lined <br> Steel or Iron | $120-150$ | 140 | 140 |
| Plastic <br> PVC or ABS | $80-150$ | 140 | 130 |
| Steel, welded and <br> seamless or <br> interior riveted | 100 |  |  |

## 555 Timer Design Equations

$\mathrm{T}=0.693\left(\mathrm{R}_{\mathrm{A}}+2 \mathrm{R}_{\mathrm{B}}\right) \mathrm{C}$
$f=\frac{1}{\mathrm{~T}}$
duty-cycle $=\frac{\left(R_{A}+R_{B}\right)}{\left(R_{A}+2 R_{B}\right)} \cdot 100 \%$
$\mathrm{T}=$ period
$f=$ frequency
$\mathrm{R}_{\mathrm{A}}=$ resistance A
$\mathrm{R}_{\mathrm{B}}=$ resistance B
C = capacitance

## Boolean Algebra

| Boolean Theorems |
| :--- |
| $X \cdot 0=0$ |
| $X \cdot 1=X$ |
| $X \cdot X=X$ |
| $X \cdot \bar{X}=0$ |
| $X+0=X$ |
| $X+1=1$ |
| $X+X=X$ |
| $X+\bar{X}=1$ |
| $\bar{X}=X$ |


| Commutative Law |
| :--- |
| $X \cdot Y=Y \cdot X$ |
| $X+Y=Y+X$ |


| Associative Law |
| :--- |
| $\mathrm{X}(\mathrm{YZ})=(\mathrm{XY}) \mathrm{Z}$ |
| $\mathrm{X}+(\mathrm{Y}+\mathrm{Z})=(\mathrm{X}+\mathrm{Y})+\mathrm{Z}$ |

Consensus Theorems
$X+\bar{X} Y=X+Y$
$X+\bar{X} \bar{Y}=X+\bar{Y}$
$\bar{X}+X Y=\bar{X}+Y$
$\bar{X}+X \bar{Y}=\bar{X}+\bar{Y}$

DeMorgan's Theorems
$\overline{X Y}=\bar{X}+\bar{Y}$
$\overline{X+Y}=\bar{X} \cdot \bar{Y}$

## Speeds and Feeds

| $N=\frac{\operatorname{cs}\left(12 \frac{\text { in }}{\text { it }}\right)}{\pi d}$ |
| :--- |
| $f_{m}=f_{t} \cdot n_{t} \cdot N$ |
| Plunge Rate $=1 / 2 \cdot f_{m}$ |
| $N=$ spindle speed (rpm) |
| $C S=$ cutting speed (in. $/ \mathrm{min}$ ) |
| $d=$ diameter (in.) |
| $f_{m}=$ feed rate (in. $/ \mathrm{min}$ ) |
| $f_{t}=$ feed (in./tooth) |
| $n_{t}=$ number of teeth |




[^0]:    GR = Gear Ratio
    $\omega_{\text {in }}=$ Angular Velocity - driver
    $\omega_{\text {out }}=$ Angular Velocity - driven
    $\mathrm{N}_{\text {in }}=$ Number of Teeth - driver
    $\mathrm{N}_{\text {out }}=$ Number of Teeth - driven
    $\mathrm{d}_{\text {in }}=$ Diameter - driver
    $\mathrm{d}_{\text {out }}=$ Diameter - driven
    $\mathrm{T}_{\text {in }}=$ Torque - driver
    $\mathrm{T}_{\text {out }}=$ Torque - driven

